

Docket: 7002/3
US Pat. Appln. No.10/511,404

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REMARKS

Claims 1-8 are now pending. Claims 9 and 10 have been canceled. Claims ~~1 and 3 are~~ the only independent claims. Claims 1, 3 and 4 have been amended herein.

First, Applicants note that a "Substitute Specification" including paragraph numbering and addressing the issue regarding the claims beginning on a new page, is submitted herewith. In addition, the specification has been amended herein to correct the number cited in paragraph [0004] of the specification (i.e., CN1068532 (Chinese Patent Number: ZL92104999.4).

The drawings were objected to under 37 CFR 1.83(a) for the reasons noted in paragraph 1 of the Action. The cancellation of Claims 9 and 10 renders this objection moot. Withdrawal of the objection is accordingly requested.

The disclosure was objected to for the reason noted in paragraph 2 of the Action. The description of Fig. 4C at paragraph [0031] has been deleted in the amendment to specification provided above. Withdrawal of the objection is accordingly requested.

As noted in paragraph 3 of the Action, claims must commence on a separate sheet. The accompanying Substitute Specification corrects this error.

Various antecedent issues were noted in paragraph 4 of the Action with respect to Claims 1, 3 and 4. The claims have been amended as necessary to address these issues.

Claims 1-10 were rejected under 35 U.S.C.112, second paragraph, as "being incomplete for omitting essential structural cooperative relationships of elements, such omission amounting to a gap between the necessary structural connections". Applicants respectfully traverse this rejection, and submit that the claims are clear in their current form. In the present case a person skilled in the art will readily be able to understand the claims after having read the description, in particular because the rear sleeve, the body, the bearing ring/the upper bearing ring and the lower bearing ring, the bearing assembly, the nut, the jaw, the front sleeve and the stopper as recited in Claims 1 and 3 are clearly illustrated in the drawings (and the claims provide reference signs linking each element to the drawings) and the detailed description of the embodiments. Furthermore, these components are known from prior art keyless chucks as noted in paragraph [0041] of the specification. Therefore, Applicant submits that a person skilled in the art would have no problem understanding the claimed arrangement.

Docket: 7002/3
US Pat. Appln. No.10/511,404

Claims 1, 5 and 7 were rejected under 35 USC 102(b) as being anticipated by US Patent 5,145,194 (Huff et al. '194) and Claims 1 and 5 were rejected as being anticipated by US Patent 5,411,275 (Huff et al. '275).

Claim 2 was rejected under 35 USC 103(a) as being unpatentable over Huff '194; Claims 3 and 10 were rejected as being unpatentable over Huff '194 in view of US Patent 5,215,317 (Jordan et al.); Claim 4 was rejected as being unpatentable over Huff '194; Claims 6 and 8 were rejected as being unpatentable over Huff '194; Claim 9 was rejected as being unpatentable over Huff '194 in view of Jordan; Claim 2 was rejected as being unpatentable over Huff '275; and Claims 6-8 were rejected as being unpatentable over Huff '275.

Applicants respectfully traverse the outstanding prior art rejections and request reconsideration based upon the foregoing amendments and the following discussion.

Independent Claim 1 is directed to a self-tightening keyless chuck for gripping a tool shank including a rear sleeve, a body, a bearing ring, a bearing assembly, a nut, a jaw, a front sleeve and a stopper. A surface of at least one of the bearing ring and the nut that faces the bearing assembly is provided with a recessed circumferential arcuate race, and a plurality of ball-shape grooves are uniformly formed in an arcuate bottom surface of the arcuate race, and wherein the front sleeve engages with the nut through radial smooth press-fit.

Independent Claim 3 further recites an *upper* bearing ring and a *lower* bearing ring, and that a surface of at least *one of* the upper bearing ring and the lower bearing ring that faces the bearing assembly is provided with a recessed circumferential arcuate race.

First, Applicants submit that there are two very important differences between Claims 1 and 3 and the cited prior art (Huff '194, Huff '275 and Jordan). First, as recited in each of Claims 1 and 3, **the race 9 on which the grooves are formed is required to be a recessed circumferential arcuate race** (see at least Fig.3A and the cross section of the race 9 in Fig.5). It is clearly shown that the race is both arcuate and recessed and runs along the whole circumference of the nut. Secondly, each of independent Claims 1 and 3 require **a plurality of ball-shaped grooves uniformly formed in the arcuate bottom surface of the arcuate race** (see at least reference sign 10 in Figs.3A, 4A and 5).

Docket: 7002/3
US Pat. Appln. No.10/511,404

In the present application, independent Claim 1 defines an embodiment in which the surface of at least one of the bearing ring 3 and the nut 5 that faces the bearing assembly 4 is provided with a recessed circumferential arcuate race 9, while independent Claim 3 defines an embodiment in which the surface of at least one of the upper bearing ring 31 and the lower bearing ring 32 that faces the bearing assembly 4 is provided with a recessed circumferential arcuate race 9. In contrast, in Huff '194 the race is flat. There is no disclosure that it is recessed or arcuate and such feature is not shown in any of the drawings.

Further, the depressions of Huff '194 are different from the grooves of the present invention. The grooves in each of independent Claims 1 and 3 are ball-shaped, and the ball-shaped grooves are formed in the arcuate bottom surface of the arcuate race. In contrast, in Huff '194 the depressions 56 are substantially rectangular. While they may have symmetrically radiused corners or non-symmetrically radiused corners, the depressions 56 are still rectangular and not ball-shaped.

Concerning Huff '275, first, as clearly shown in Fig.3 and illustrated in column 4, lines 19-40, the surface of cam ring 80 has four channels 84 (corresponding to the race of the present invention as mentioned by the Examiner) that are spaced from one,another around the cam ring, and each channel has a inclined bottom with a shallower end and a deeper end. In contrast, the race 9 of Applicant's invention runs continuously along the whole circumference of, for example, the nut, and has a uniform depth. Further, although it is mentioned in Huff '275 that the shallower end 84b can be slightly deeper than the immediately adjacent portion of the channel, the purpose of this is merely to retain the ball at the shallower end of the channel. In view of this purpose, a channel 84 can only have one so-called "ball-shaped groove" at a certain location. Instead, the ball-shaped grooves of the present claimed invention can be uniformly formed in bottom surface of the race with more free arrangement. Based on at least the foregoing distinctions, Applicant respectfully submits that the structure of the present invention as claimed herein is distinguished over that of Huff '275.

Jordan discloses a substantially same cam ring as Huff '275 with respect to its channels.

Docket: 7002/3
US Pat. Appln. No.10/511,404

Applicant submits that the based upon the two foregoing distinguishing features, the present invention, as claimed herein, has the following advantages: during operation, the balls of the bearing assembly enter the ball-shaped grooves of the bearing ring or the nut and are locked therein. Alternatively, the balls of the bearing assembly enter the ball-shaped grooves of the upper bearing ring or the lower bearing ring and are locked therein. Thus, the nut is less susceptible to loosening in operation, so that the chuck maintains a great gripping force. Further, due to the use of the recessed circumferential arcuate race, the balls of the bearing assembly are constantly constrained within the arcuate race. This will ensure the centering of the members under force, so that the three jaws of the chuck are subjected to a uniform force to thereby increase the gripping force.

In view of the foregoing claim amendments and discussion, Applicant respectfully submits that each of independent Claims 1 and 3 is patentable over any combination of teachings of Huff '194, Huff '275 and Jordan based upon each of the foregoing novel features. Further, Claims 1 and 3 are inventive as their novel features are advantageous and could not be derived in an obvious manner from Huff '194, Huff '275 and Jordan or any combination thereof.

In light of the following discussion, the rejection is respectfully traversed and reconsideration is requested.

The application is believed to be in condition for allowance. Prompt review and issuance is accordingly requested. Should the Examiner be of the view that an interview would expedite consideration of this Amendment or of the application at large, request is made that the Examiner telephone the Applicants' undersigned attorney at (908) 518-7700 in order that any outstanding issues be resolved.

Docket: 7002/3
US Pat. Appln. No.10/511,404

If there are any fees due and owing in respect to this amendment, the Examiner is authorized to charge such fees to deposit account number 50-1047.

Respectfully submitted,



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I hereby certify that this correspondence and any document referenced herein is being sent to the United States Patent and Trademark office via Facsimile to: 571-273-8300 on 12/13/06

Date 12/13/06Sig: Karin L. Williams
Name: Karin L. Williams**IN THE UNITED STATES PATENT AND TRADEMARK OFFICE**

Applicant: Xianming Tian

Serial No.: 10/511,404

Filed: 10/12/04

Title: SELF-TIGHTENING KEYLESS CHUCK

Art Unit: 3722

Examiner: Michael Talbot

Confirmation Number: 9904

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Via Facsimile to:
571-273-8300
Commissioner for Patents
P.O. Box 1450
Alexandria, VA 22313-1450

SUBMISSION OF SUBSTITUTE SPECIFICATION UNDER 37 CFR 1.125(b)

Sir:

Applicant hereby submits a substitute specification in accordance with 37 CFR 1.125(b):

This substitute specification contains no new matter. This substitute specification is filed only to include paragraph numbering on the specification pages and to begin the claims on a separate page as required.

In addition, as required under 37 CFR 1.125(b), a marked-up copy of the substitute specification showing the changes made to the specification of record is also submitted herewith.

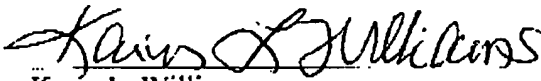
An Amendment and Response to Office Action is filed herewith. The Examiner is

1

invited to telephone the Applicant's attorney at (908) 518-7700 in order that any outstanding issues be resolved.

If there are any fees due and owing in respect to this submission, the Examiner is authorized to charge such fees to deposit account number 50-1047.

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Docket. No. 7002/3
Substitute Spec - Marked-Up Version

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SELF-TIGHTENING KEYLESS CHUCK

FIELD OF THE INVENTION

[0001] The present application relates to a keyless chuck and, more particularly, to a self-tightening keyless chuck. Generally, an electric percussive drill is capable of reciprocating, percussive vibration as well as rotary motion. Therefore, the present invention particularly provides a self-tightening keyless chuck for maintaining a secure grip on such a tool in percussive motion.

BACKGROUND ART

[0002] A variety of keyless chucks have been developed so far. A conventional chuck generally comprises a rear sleeve, a body, a bearing ring, a bearing, a nut, a jaw, a front sleeve, a stopper and so on. The jaw extends/retracts under the rotation of the nut and the engagement of conical threads, and the nut is rotated by movement of the front sleeve. Such a chuck is keyless if the front sleeve is rotated by hand. The chuck may be attached to a driving shaft of a drill by means of threads or a conical bore.

[0003] A conventional keyed or keyless chuck has the disadvantage that because the nut and the jaw are secured only by rotation of threads, the jaw tends to loosen its grip on a tool due to axial percussive vibrations transmitted through the chuck and the tool during operation. This will cause the tool to skid, thus resulting in a loss of working efficiency and a loss of safety.

[0004] A conventional chuck is disclosed in Chinese Patent Publication CN1040732C in which the anti-percussion of the chuck is achieved by forming spokewise grooves in the surfaces of two members that act as bearing races, the opposing sides of the grooves being provided with round angles of different radii. The depth of the grooves has a great influence on the anti-percussion ability of the chuck as well as on the release of the chuck. If the grooves are too deep, the chuck will not be released after use. If the grooves are too shallow, the chuck will have a poor anti-percussion performance. Therefore, in order to keep a uniform depth of the grooves in all members to thereby ensure the anti-percussion ability of all chucks, it is necessary to frequently adjust the machine for manufacturing the members. This, however, will affect the efficiency of production and can result in an increase of defective products. To overcome this drawback, it is necessary to provide a pressing machine with a higher pressing accuracy or larger tonnage, which however will increase the cost of production.

[0005] Furthermore, the front sleeve of a conventional keyless chuck is generally

Docket. No. 7002/3

Substitute Spec - Marked-Up Version

made of plastic. The chuck generally has a configuration in which a steel ring is embedded in an inner side of the plastic front sleeve, and two nut halves that are placed in a circular groove in the body of the chuck are press-fit into the hole of the steel ring embedded in the inner side of the plastic front sleeve, thereby securing the two nut halves in the circular groove in the body. This configuration, however, is complicated in structure and manufacturing process, and the front sleeve of the chuck which has a low strength is susceptible to damage.

SUMMARY OF THE INVENTION

[0006] To overcome the aforementioned disadvantages and drawbacks in the prior art, the present invention provides a safe and simple self-tightening keyless chuck, which has a locking function and improves the strength of a front sleeve and which is less susceptible to loosening when used for a percussive tool.

[0007] In order to fulfill the above object, the present invention employs the following technical solution.

[0008] The invention provides a self-tightening keyless chuck for gripping a tool shank, comprising a rear sleeve, a body, a bearing ring, a bearing assembly, a nut, a jaw, a front sleeve and a stopper, wherein the surface of at least one of the bearing ring and the nut that faces the bearing assembly is provided with a recessed circumferential arcuate race, and a plurality of ball-shape grooves are uniformly formed in an arcuate bottom surface of the arcuate race, and wherein the front sleeve engages with the nut through radial smooth press-fit.

[0009] The nut is provided with two angularly-cut split notches that are symmetrically positioned on both ends of a diameter of the nut, the angularly-cut split notches having a cut-out angle between 30 and 75°.

[0010] Alternatively, the invention provides a self-tightening keyless chuck for gripping a tool shank, comprising: a rear sleeve, a body, an upper bearing ring, a bearing assembly, a lower bearing ring, a nut, a jaw, a front sleeve and a stopper, wherein the surface of at least one of the upper and lower bearing rings that faces the bearing assembly is provided with a recessed circumferential arcuate race, and a plurality of ball-shape grooves are uniformly formed in the arcuate bottom surface of the arcuate race, and wherein the front sleeve engages with the nut through radial smooth press-fit.

[0011] In this self-tightening keyless chuck, the nut is provided with two angularly-cut split notches that are symmetrically positioned on both ends of a diameter of the nut, the angularly-cut split notches having a cut-out angle between 30 and 75°. Further, two parallel cut-out surfaces are symmetrically formed on the outer peripheral surface of the nut, which extend downwards in an axial direction from the front end surface of the nut by one third of the thickness of the nut and are

Docket. No. 7002/3

Substitute Spec - Marked-Up Version

located at an angle of 90° relative to the positions of angularly-cut split notches.

[0012] The front sleeve is press cast using a press-casting metal or a powdered metallurgic material.

[0013] In the above two self-tightening keyless chucks, the recessed circumferential arcuate race has an arc radius R of from 1.5 to 2.5 millimeters.

[0014] The recessed circumferential arcuate race has a depth H from 0.05 to 0.30 millimeters.

[0015] The number of ball-shaped grooves ranges from 20 to 80.

[0016] The ball-shaped grooves have a spherical radius r from 1.0 to 2.0 millimeters.

[0017] The ball-shaped grooves have a depth h from 0.01 to 0.10 millimeters.

[0018] In the alternative self-tightening keyless chucks for gripping a tool shank, the upper bearing ring and the lower bearing ring may have the same external dimensions and the same surface structure.

[0019] The upper bearing ring and the lower bearing ring may also have different external dimensions and different surface structures, and their differences include the inside diameter and the thickness of the bearing rings.

[0020] During operation, the balls of the bearing assembly enter the ball-shaped grooves of the bearing ring or the nut and are locked therein. Alternatively, the balls of the bearing assembly enter the ball-shaped grooves of the upper bearing ring or the lower bearing ring and are locked therein. Thus, the nut is less susceptible to loosening in operation, so that the chuck maintains a great gripping force.

[0021] The present invention has an advantage that the depth of the ball-shaped grooves which is essential to the anti-percussion ability of the chuck is determined by a mould. That is, the optimum distance (i.e., depth of the ball-shaped grooves) between the ball-shaped grooves and the arcuate bottom surface of the recessed circumferential arcuate race has been determined during the manufacturing of the mould. In this way, the pressing of the members has nothing to do with the adjustment or accuracy of the machine for manufacturing the members, whereby the depth of the ball-shaped grooves in all members can be constantly maintained within an optimum range to ensure the anti-percussion performance of all chucks.

[0022] The present invention has another advantage that due to the use of the recessed arcuate race, the balls of the bearing assembly are constantly constrained within the arcuate race. This will ensure the centering of the members under force, so that the three jaws of the chuck are subjected to a uniform force to thereby increase

Docket. No. 7002/3
Substitute Spec - Marked-Up Version
the gripping force.

[0023] The present invention has still another advantage that because the front sleeve is press cast using a press-casting metal or a powdered metallurgic material, the front sleeve has an increased strength and can be directly engaged with the nut through press-fit to securely transmit a torque.

[0024] By using the above configurations, the present invention overcomes the problem that a conventional chuck either does not have a locking means or has a locking means with unsatisfactory performance. The chuck according to the present invention can be operated in a quick, reliable, safe and convenient way. It provides such advantages as a reasonable structure, a convenient assembly, a stable performance and a reliable operation. The present invention can be used in all kinds of keyed or keyless chucks.

BRIEF DESCRIPTION OF THE DRAWINGS

[0025] Fig. 1 is a front view of a self-tightening keyless chuck according to the present invention.

[0026] Fig. 2 is an axial, longitudinal sectional view according to an embodiment of the present invention shown in Fig. 1.

[0027] Fig. 3A is a front view of a bearing ring according to an embodiment of the present invention.

[0028] Fig. 3B is a perspective view of a bearing ring according to an embodiment of the present invention.

[0029] Fig. 4A is a front view of a nut according to an embodiment of the present invention.

[0030] Fig. 4B is a perspective view of a nut according to an embodiment of the present invention.

[0031] Fig. 4C is a rear view of a nut according to an embodiment of the present invention.

[0032] Fig. 5 is a partially enlarged cross-sectional view of a ball-shaped groove in an arcuate bottom surface of an arcuate race used in common by a bearing ring and a nut according to an embodiment of the present invention, i.e., an enlarged cross sectional view taken along line k-k in Figs. 3A and 4A.

[0033] Fig. 6 is an axial, longitudinal sectional view according to another embodiment of the present invention shown in Fig. 1.

Docket No. 7002/3
Substitute Spec - Marked-Up Version

[0034] Fig. 7A is a partially enlarged view in which the balls of a bearing assembly are outside ball-shaped grooves (i.e., the balls are in a non-self-tightening state).

[0035] Fig. 7B is a partially enlarged view in which the balls of a bearing assembly are inside ball-shaped grooves (i.e., the balls are in a self-tightening state).

[0036] Fig. 8A is a front view of a nut according to another embodiment of the present invention.

[0037] Fig. 8B is a perspective view of a nut according to another embodiment of the present invention.

[0038] Fig. 8C is a rear view of a nut according to another embodiment of the present invention.

DETAILED DESCRIPTION OF THE EMBODIMENTS

[0039] A detailed description of the present invention is provided with reference to the drawings and embodiments below.

[0040] Fig. 1 is a front view of a keyless chuck manufactured according to the preferred embodiment of the present invention.

[0041] Fig. 2 is an axial, longitudinal sectional view of a self-tightening chuck according to an embodiment of the present invention, wherein the chuck comprises a rear sleeve 1, a body 2, a bearing ring 3, a bearing assembly 4, a nut 5, a jaw 6, a front sleeve 7 and a stopper 8. The relations of these elements are known in the art, and hence a detailed description thereof is omitted.

[0042] Referring to Figs. 3, 4 and 5, the bearing ring 3 has a race surface that is provided with a circumferential arcuate race 9 with a fixed depth. When viewed from the cross section along line k-k, the circumferential arcuate race 9 has an arc radius R of 2.0 millimeters and a depth H of 0.15 millimeters. The circumferential arcuate race 9 has an arcuate bottom surface that is provided with 40 ball-shaped grooves 10, which have a spherical radius r of 1.5 millimeters and a depth h of 0.05 millimeters. Further, the surface of the nut 5 that acts as the other race of the bearing assembly 4 is also provided with an circumferential arcuate race 9 and ball-shaped grooves 10 that are identical in shape and number to those formed on the bearing ring 3. During operation, as shown in Fig. 7, the jaw grips a tool shank as the nut 5 rotates, and the balls of the bearing assembly 4 goes into/out of the ball-shaped grooves. Fig. 7A shows a non-self-tightening state of the balls of the bearing assembly 4 in which the balls are outside the ball-shaped grooves. When the nut 5 is finally secured on the body 2, the balls of the bearing assembly 4 enter into the ball-shaped grooves 10 in the upper or lower race and are locked therein. Fig. 7B

Docket. No. 7002/3

Substitute Spec - Marked-Up Version

shows a self-tightening state of the balls of the bearing assembly 4 in which the balls are in the ball-shaped grooves. Thus, the percussive vibrations transmitted through the body 2 of the chuck are unlikely to overcome the difference between the high and low energy states of the balls of the bearing assembly 4, the low energy state corresponding to the location of the balls in the ball-shaped grooves. Depending upon the specific application of the chuck and the expected impact loads during operation, the choice of having ball-shaped grooves in either or both of the race surfaces of the nut 5 and the bearing ring 3 and the number and depth of the ball-shaped grooves can be optimized.

[0043] Fig. 6 is an axial, longitudinal sectional view of a self-tightening chuck according to another embodiment of the present invention, wherein the chuck comprises a rear sleeve 1, a body 2, an upper bearing ring 31, a bearing assembly 4, a lower bearing ring 32, a nut 5, a jaw 6, a front sleeve 7 and a stopper 8. In this embodiment, the lower bearing ring 32 is used as the other race of the balls of the bearing assembly 4 in place of the nut 5 in the previous embodiment. The upper bearing ring 31 and the lower bearing ring 32 may have the same external dimensions. The surface of at least one of the upper and lower bearing rings that faces the bearing assembly 4 is provided with a recessed circumferential arcuate race 9, and a plurality of ball-shape grooves 10 are uniformly formed in an arcuate bottom surface of the arcuate race 9. The chuck in this embodiment operates according to the same working principle as that in the previous embodiment, and hence a detailed description thereof is omitted.

[0044] During assembly, the metal front sleeve 7 engages, through press-fit, with the nut 5 that is placed in a circular groove in the body 2 and split into two halves by split notches 11. Thanks to the strength and the elastic/plastic deformability of the metal material per se, the metal front sleeve 7 is securely connected to the nut 5 through press-fit, thus ensuring transmission of an input torque. Before press-fit, the metal sleeve 7 is surface-treated by electrophoresis painting, static plastic spraying or anodized processing so that its surface meets different requirements in color.

Docket. No. 7002/3

Substitute Spec - Marked-Up Version

What is claimed is:

1. A self-tightening keyless chuck for gripping a tool shank, comprising:

a rear sleeve (1);
a body (2);
a bearing ring (3);
a bearing assembly (4);
a nut (5);
a jaw (6);
a front sleeve (7); and
a stopper (8),

wherein the surface of at least one of the bearing ring (3) and the nut (5) that faces the bearing assembly (4) is provided with a recessed circumferential arcuate race (9), and a plurality of ball-shape grooves (10) are uniformly formed in an arcuate bottom surface of the arcuate race (9),
and wherein the front sleeve (7) engages with the nut (5) through radial smooth press-fit.

2. The self-tightening keyless chuck according to claim 1, wherein the nut (5) is provided with two angularly-cut split notches (11) that are symmetrically positioned on both ends of a diameter of the nut, the angularly-cut split notches (11) having a cut-out angle between 30 and 75°.

3. A self-tightening keyless chuck for gripping a tool shank, comprising:

a rear sleeve (1);
a body (2);
an upper bearing ring (31);
a bearing assembly (4);
a lower bearing ring (32);
a nut (5);
a jaw (6);
a front sleeve (7); and
a stopper (8),

wherein the surface of at least one of the upper bearing ring (31) and the lower bearing ring (32) that faces the bearing assembly (4) is provided with a recessed circumferential arcuate race (9), and a plurality of ball-shape grooves (10) are uniformly formed in an arcuate bottom surface of the arcuate race (9),
and wherein the front sleeve (7) engages with the nut (5) through radial smooth press-fit.

4. The self-tightening keyless chuck according to claim 3, wherein the nut (5) is

Docket. No. 7002/3

Substitute Spec - Marked-Up Version

provided with two angularly-cut split notches (11) that are symmetrically positioned on both ends of a diameter of the nut, the angularly-cut split notches (11) having a cut-out angle between 30 and 75° ,

and wherein two parallel cut-out surfaces (13) are symmetrically formed on the outer peripheral surface of the nut (5), which extend downwards in an axial direction from the front end surface of the nut (5) by one third of the thickness of the nut and are located at an angle of 90° relative to the positions of angularly-cut split notches (11).

5. The self-tightening keyless chuck according to claim 1 or 3, wherein the front sleeve (7) is press cast using a press-casting metal or a powdered metallurgic material.

6. The self-tightening keyless chuck according to claim 1 or 3, wherein the recessed circumferential arcuate race (9) has an arc radius R from 1.5 to 2.5 millimeters and a depth H from 0.05 to 0.30 millimeters.

7. The self-tightening keyless chuck according to claim 1 or 3, wherein the number of ball-shaped grooves (10) ranges from 20 to 80.

8. The self-tightening keyless chuck according to claim 1 or 3, wherein the ball-shaped grooves (10) have a spherical radius r from 1.0 to 2.0 millimeters and a depth h from 0.01 to 0.10 millimeters.

9. The self-tightening keyless chuck according to claim 3, wherein the upper bearing ring (31) and the lower bearing ring (32) have the same external dimensions and the same surface structure.

10. The self-tightening keyless chuck according to claim 3, wherein the upper bearing ring (31) and the lower bearing ring (32) have different external dimensions and different surface structures, their differences including the inside diameter and the thickness of the bearing rings.

Docket. No. 7002/3
Substitute Spec - Marked-Up Version

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ABSTRACT

The present invention relates to a self-tightening keyless chuck to enable electric percussive drills to grip a tool shank. The chuck includes a rear sleeve, a body, a bearing ring, a bearing assembly, a nut, a jaw, a front sleeve and a stopper and so on. The bearing assembly utilizing a ball bearing is positioned with a circumferential arcuate race with a fixed depth, and the bottom surfaces of said circumferential arcuate race are provided with a plurality of ball-shape grooves. During the chuck grips a tool shank, the balls of said ball-bearing enter into the ball-shape grooves and are locked there, so as to prevent the tool from loosening due to the percussion vibration. In addition, the front sleeve engages with said nut through radial smooth press-fit to securely transmit a torque.